

Oldroyd 关节液的润滑效应

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摘要 本文选取 Oldroyd 流体模拟人体关节液, 分别用楔形滑缝、指数形滑缝模拟人体膝关节, 讨论关节液在滑动关节中的润滑效应. 用摄动法求解, 以 Deborah 数作为摄动参数, 求到二阶近似解, 并引用实验得到的三类人 (普通青年人、老年人和骨关节病人) 的粘弹性参量进行计算和分析, 得到各类人粘弹性参量的润滑效应, 为病理诊断研究提供依据.

关键词 Oldroyd 流体; 关节液; 楔形滑缝; 指数形滑缝; 膝关节; 润滑效应; 摄动法.

一、动关节和关节液概述及模式

人体内有許多关节, 比如髋、膝、肩和指关节等等, 它们是在骨架生长时期形成的, 其主要功能是允许骨头之间作适当程度的光滑的相对运动, 并承受一定的载荷. 图1 是人体膝关节略图. 如图所示, 包围两骨的纤维状外壳为关节膜, 关节膜的内壁为滑膜. 覆盖在骨端一层较硬的薄层为关节软骨. 由滑膜和关节软骨围成的封闭小室为关节腔, 里面有小量的滑液. 在关节腔周围的韧带、腱及其它软组织, 在关节活动时保持关节的适当姿态, 使关节活动稳定. 人体膝关节的主要作用是在人行走或奔跑时, 使两腿伸屈, 因而它能在各种速度及急骤变化承载力下作出相当复杂的运动, 通常它能承受五倍于人体的重量, 并能健康地工作七、八十年.

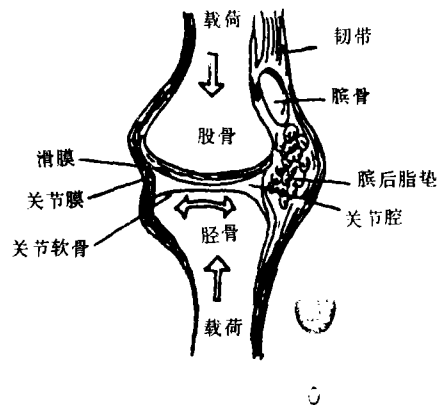


图1 人体膝关节略图

作为动关节力学性能的初步研究, 暂不考虑关节软骨组织对滑液的渗吸及一些生理现象, 而将关节理想化, 选取楔形滑缝 (如图2 所示) 和指数形滑缝 (如图3 所示) 模拟人体膝关节, 研究关节活动时滑液产生的力学性能. 由于关节的滑缝宽度 $h(x)$ 远小于缝长 L , 所以可用润滑近似^[1-2].

密闭在关节腔内的滑液通常是透明、无色或微黄的液体, 大约0.2毫升. 这种滑液是血浆与透明质酸蛋白复合体 (HAP) 组成的一种透析液, 它是一种无分岔的多糖大分子, 它基本的二聚物是由葡萄糖醛酸与氨基葡萄糖交联组成的双糖酶, HAP 复合体的平均分子量大约是二百万, 在溶液里这些大分子缠结成球形或椭圆形. 许多实验表明滑液具有拟塑性、弹

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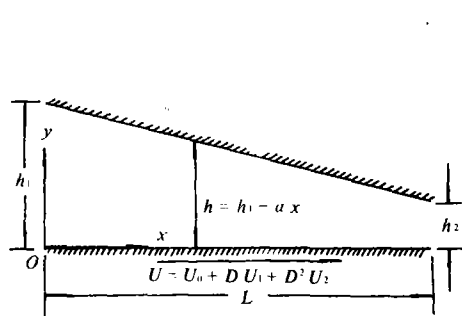


图2 楔形滑缝

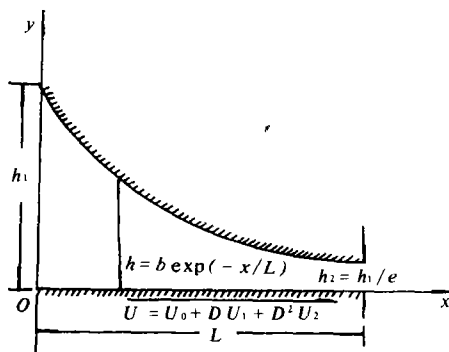


图3 指数形滑缝

性及触变性等非牛顿性质,一些研究者还分别测出了它的表观粘度、复动力模量、法向应力差以及应力松弛函数^[1],在我们的研究中,选取四参量的Oldroyd流体模拟这种滑液,由于Oldroyd流体具有表现应力过量的弹性性能和剪切变稀的特性^[4],一些作者也认为Oldroyd流体描述复杂流动较好^[5]。

二、基本方程

在润滑近似下,图2、图3所示滑缝中液体的速度分布为

$$\underline{u} = (u, 0, 0) \quad (1)$$

四参量Oldroyd流体的本构方程为

$$\underline{T} + \lambda_1 \frac{\delta \underline{T}}{\delta t} + \frac{\mu_0}{2} (\text{tr} \underline{T}) \underline{A}_1 = \eta_0 \left(1 + \lambda_2 \frac{\delta}{\delta} \right) \underline{A}_1 \quad (2)$$

式中 $\underline{T}$ 为偏应力张量, $\underline{A}_1$ 为一阶Rivlin-Ericksen张量, λ_1 为松弛时间, λ_2 为推进时间, η_0 为零剪切粘度, μ_0 是具有时间量纲的非线性粘度参量, $\frac{\delta}{\delta t}$ 为Oldroyd导数,对于任一个二阶张量 B^{ij} ,有

$$\frac{\delta}{\delta t} B^{ij} = \frac{\partial B^{ij}}{\partial t} + v^k \frac{\partial B^{ij}}{\partial x^k} - \frac{\partial v^i}{\partial x^k} B^{kj} - \frac{\partial v^j}{\partial x^k} B^{ik} \quad (3)$$

由(1)式(2)式算出偏应力张量 $\underline{T}$ 的物理分量为

$$\left. \begin{aligned} T_{xx} + \lambda_1 \left(u \frac{\partial T_{xx}}{\partial x} - 2 \frac{\partial u}{\partial y} T_{xy} \right) &= -2 \eta_0 \lambda_2 \left(\frac{\partial u}{\partial y} \right)^2 \\ T_{xx} + \lambda_1 \left(u \frac{\partial T_{xx}}{\partial x} - \frac{\partial u}{\partial y} T_{xy} \right) + \frac{\mu_0}{2} (T_{xx} + T_{yy} + T_{zz}) \frac{\partial u}{\partial y} &= \eta_0 \frac{\partial u}{\partial y} \\ T_{xx} + \lambda_1 u \frac{\partial T_{xx}}{\partial x} &= 0 \\ T_{zz} + \lambda_1 \left(u \frac{\partial T_{zz}}{\partial x} - \frac{\partial u}{\partial y} T_{yz} \right) &= 0 \end{aligned} \right\} \quad (4)$$

$$T_{xz} + \lambda_1 u \frac{\partial T_{xz}}{\partial x} = 0$$

$$T_{zz} + \lambda_1 u \frac{\partial T_{zz}}{\partial x} = 0$$

取 Deborah 数

$$D = \frac{\lambda_1}{L/U} \quad (5)$$

将各量无量纲化

$$x = L \bar{x}, y = L \bar{y}, u = U \bar{u}, T_{xz} = \frac{\eta_0 U}{L} \bar{T}_{xz}, \mu_0 = \lambda_1 \bar{\mu}_0, \lambda_2 = \lambda_1 \bar{\lambda}_2 \quad (6)$$

$$p = \frac{\eta_0 U}{L} \bar{p}, \beta_1 = \frac{U_1}{U_0}, \beta_2 = \frac{U_2}{U_0}$$

可以看出 (4) 式中仅有的非零偏应力分量 $\bar{T}_{xz}$ 与 $\bar{T}_{zx}$, 令 $\sigma = \bar{T}_{xz}$, $\tau = \bar{T}_{zx}$, 并应用润滑近似, 则由 (4) 式可以得到

$$\left. \begin{aligned} \sigma - 2D \dot{\gamma} \tau &= -2D \bar{\lambda}_2 \dot{\gamma}^2 \\ \tau + \frac{\bar{\mu}_0}{2} D \sigma \dot{\gamma} &= \dot{\gamma} \end{aligned} \right\} \quad (7)$$

式中 $\dot{\gamma} = \frac{\partial \bar{u}}{\partial \bar{y}}$.

在润滑近似下, 运动方程为

$$\frac{\partial \bar{p}}{\partial \bar{x}} = \frac{\partial \tau}{\partial \bar{y}}, \quad \frac{\partial \bar{p}}{\partial \bar{y}} = 0 \quad (8)$$

边界条件是

$$\bar{u}(\bar{h}) = 0, \bar{u}'(0) = 1 + \beta_1 D + \beta_2 D^2, \bar{p}(0) = \bar{p}(1) = 0 \quad (9)$$

由于本文后面将讨论牛顿流体和非牛顿流体边界条件的差别, 故在 (9) 式中存在对 D 和 D^2 的依赖性.

将 (7) 式代入 (8) 式之后, 得到 $\bar{u}$ 的一个非线性方程, 我们用摄动法求其解, 将 $\bar{u}$ 和 $\bar{p}$ 对 D 展开

$$\bar{u} = \bar{u}_0 + D \bar{u}_1 + D^2 \bar{u}_2, \bar{p} = \bar{p}_0 + D \bar{p}_1 + D^2 \bar{p}_2 \quad (10)$$

从而得到零阶近似方程为

$$\tau_0 = \dot{\gamma}_0, \sigma_0 = 0; \quad \frac{\partial \bar{p}_0}{\partial \bar{x}} = \dot{\gamma}_0 \quad (11)$$

一阶近似方程为

$$\tau_1 = \dot{\gamma}_1, \sigma_1 = 2(1 - \bar{\lambda}_2) \dot{\gamma}_0^2, \quad \frac{\partial \bar{p}_1}{\partial \bar{x}} = \dot{\gamma}_1 \quad (12)$$

和二阶近似方程为

$$\left. \begin{aligned} \tau_2 &= \dot{\gamma}_2 - \bar{\mu}_0(1 - \bar{\lambda}_2)\dot{\gamma}_0^3, \quad \sigma_2 = 4(1 - \bar{\lambda}_2)\dot{\gamma}_0\dot{\gamma}_1, \\ \frac{\partial \bar{p}_2}{\partial x} &= \dot{\gamma}_2 - 3\bar{\mu}_0(1 - \bar{\lambda}_2)\dot{\gamma}_0^2\dot{\gamma}_0 \end{aligned} \right\} \quad (13)$$

三、基本解

1. 对于楔形滑缝

零阶解: 在边界条件(9)下积分(11)式, 得到

$$\bar{u}_0 = \frac{1}{2}G_0\bar{y}^2 - \left(\frac{1}{h} + \frac{G_0}{2}\bar{h}\right)\bar{y} + 1 \quad (14)$$

式中

$$G_0 = \frac{6}{h^2} - \frac{12}{h^3}Q^{(0)}, \quad Q^{(0)} = \frac{\bar{h}_1\bar{h}_2}{h_1 + h_2} \quad (15)$$

从而算出

$$\tau_0 = G_0\bar{y}\left(1 - \frac{\bar{h}}{2}\right) - \frac{1}{h} \quad (16)$$

故可算得总支撑力

$$W_0 = \int_0^L \sigma_0|_{y=0} dx = \bar{p}_0 L + \frac{6\eta_0 U}{a^2} \left(\ln H - 2 \frac{H-1}{H+1} \right) \quad (17)$$

及总摩阻力

$$R_0 = \int_0^L \tau_0|_{y=0} dx = \frac{2\eta_0 U}{a} \left(-2 \ln H + 3 \frac{H-1}{H+1} \right) \quad (18)$$

式中 $H = h_1/h_2$.

一阶解: 用类似方法可算出

$$\left. \begin{aligned} \bar{u}_1 &= \frac{1}{a}G_1\bar{y}^2 - \left(\frac{\beta_1}{h} + \frac{1}{a}G_1\bar{h}\right)\bar{y} + \beta_1 \\ G_1 &= \frac{6\beta_1}{h^2} - \frac{12}{h^3}Q^{(1)}, \quad Q^{(1)} = \frac{\beta_1\bar{h}_1\bar{h}_2}{h_1 + h_2} \\ \tau_1 &= G_1\bar{y} - \left(\frac{\beta_1}{h} + \frac{1}{2}G_1\bar{h}\right) \end{aligned} \right\} \quad (19)$$

及总支撑力

$$W_1 = \frac{6\eta_0 U}{a^2} \frac{\beta_1 D}{H+1} \left(\ln H - 2 \frac{H-1}{H+1} \right) \quad (20)$$

和总摩阻力

$$R_1 = \frac{2\eta_0 U}{a} \frac{\beta_1 D}{H+1} \left(-2 \ln H + 3 \frac{H-1}{H+1} \right) \quad (21)$$

二阶解:

$$\bar{u}_2 = \frac{G^2}{2}\bar{y}^2 + 3\bar{\mu}_0(1 - \bar{\lambda}_2)G_0 \left(\frac{G_0}{12}\bar{y}^4 - \frac{G_0}{3}\frac{\bar{y}^3}{h} - \frac{G_0^2}{6}\bar{h}\bar{y}^3 + \frac{G_0}{2}\bar{y} + \frac{G_0^2}{8}\bar{y} + \frac{G_0^2}{8}\bar{h}^2\bar{y}^2 + \frac{\bar{y}^2}{2\bar{h}^2} \right)$$

$$\left. \begin{aligned} & - \left[\frac{1}{h} + \frac{G_0}{2} \bar{h} + 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) G_0 \left(\frac{G_0^3}{12} \bar{h}^3 + \frac{1}{2\bar{h}} + \frac{G_0}{6} \bar{h} - \frac{G_0^3}{24} \bar{h}^3 \right) \right] \bar{y} + 1 \\ G_2 = & - \frac{12}{h^3} Q_x^{(2)} + \frac{6}{h^2} - 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) G_0 \left(\frac{G_0^3}{20} \bar{h}^2 + \frac{1}{h^2} \right) \\ Q_x^{(2)} = & \frac{\bar{h}_1^2 \bar{h}_2^2}{2(\bar{h}_1^2 - \bar{h}_2^2)} \left\{ \frac{2(\bar{h}_1 - \bar{h}_2)}{\bar{h}_1 \bar{h}_2} + \bar{\mu}_0 (1 - \bar{\lambda}_2) \left[-\frac{28}{5} \frac{\bar{h}_1^3 - \bar{h}_2^3}{\bar{h}_1^3 \bar{h}_2^3} + \left(3 + \frac{81}{5} Q_x^{(0)} \frac{\bar{h}_1^4 - \bar{h}_2^4}{\bar{h}_1^4 \bar{h}_2^4} \right. \right. \right. \\ & \left. \left. \left. - \frac{648}{25} Q_x^{(0)^2} \frac{\bar{h}_1^5 - \bar{h}_2^5}{\bar{h}_1^5 \bar{h}_2^5} + \frac{72}{5} Q_x^{(0)^3} \frac{\bar{h}_1^6 - \bar{h}_2^6}{\bar{h}_1^6 \bar{h}_2^6} \right) \right] \right\} \end{aligned} \right\} \quad (22)$$

总支撑力

$$\begin{aligned} W_2 = & - \frac{3 \eta_0 U D^2}{\alpha^2} \left\{ \frac{2 Q_x^{(2)}}{h_1} \left(\frac{1}{H} - 1 \right) + 2 Q_x^{(2)} \beta_2 \frac{H-1}{h_1} + 2 \left(1 - \frac{1}{H} \right) - 2 \ln H \right. \\ & + \bar{\mu}_0 (1 - \bar{\lambda}_2) \left[\frac{16}{5} \frac{1-H}{H \bar{h}_1^2} + \frac{9}{5} \frac{H^2-1}{\bar{h}_1^2} - \frac{27}{5} Q_x^{(0)} \frac{H^3-1}{\bar{h}_1^3-1} - \frac{81}{5} Q_x^{(0)} \frac{1-H}{H \bar{h}_1^3} \right. \\ & + \frac{648}{25} Q_x^{(0)^2} \frac{1-H}{H \bar{h}_1^4} + \frac{162}{25} Q_x^{(0)^2} \frac{H^4-1}{\bar{h}_1^4} - \frac{72}{25} Q_x^{(0)^3} \frac{H^5-1}{\bar{h}_1^5} - \frac{72}{5} Q_x^{(0)^3} \frac{1-H}{H \bar{h}_1^5} \\ & \left. \left. + \frac{2(1-H)}{H \bar{h}_1^2} + \frac{H^2-1}{\bar{h}_1^2} + \frac{1-H^3}{\bar{h}_1^3} - \frac{3(1-H)}{H \bar{h}_1^3} \right] \right\} \end{aligned} \quad (23)$$

和总摩阻力

$$\begin{aligned} R_2 = & \frac{D^2}{\alpha} \left\{ -4 \beta_2 \ln H + 6 Q_x^{(2)} \frac{H-1}{h_1} + 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) \left[\frac{53}{15} \frac{H^2-1}{\bar{h}_1^2} \right. \right. \\ & \left. \left. - \frac{252}{15} Q_x^{(0)} \frac{H^3-1}{\bar{h}_1^3} + \frac{96}{5} Q_x^{(0)^2} \frac{H^4-1}{\bar{h}_1^4} - \frac{216}{25} Q_x^{(0)^3} \frac{H^5-1}{\bar{h}_1^5} \right] \right\} \end{aligned} \quad (24)$$

2. 对于指数形滑缝

同样可求得零阶解:

$$\left. \begin{aligned} \bar{u}_0 = & \frac{1}{2} G_0 \bar{y}^2 - \left(\frac{1}{h} + \frac{G_0}{2} \bar{h} \right) \bar{y} + 1 \\ G_0 = & \frac{6}{h^2} - \frac{12}{h^3} Q_x^{(0)}, \quad Q_x^{(0)} = \frac{3}{4} \frac{\bar{h}_1^2 + \bar{h}_2^2}{\bar{h}_1^2 + \bar{h}_1 \bar{h}_2 + \bar{h}_2^2} \end{aligned} \right\} \quad (25)$$

总支撑力

$$W_0 = \bar{p}_0 L + 6 L^2 \eta_0 U \left[-\frac{\ln H}{2 \bar{h}_1^2} + \frac{2}{3} \frac{\ln H}{\bar{h}_1^3} + \frac{1}{4} \frac{H^2-1}{\bar{h}_1^2} + \frac{2}{9} \frac{1-H^3}{\bar{h}_1^3} \right] \quad (26)$$

和总摩阻力

$$R_0 = L \eta_0 U \left[4 \frac{1-H}{\bar{h}_1} + 3 Q_x^{(0)} \frac{H^2-1}{\bar{h}_1^2} \right] \quad (27)$$

一阶解:

$$\left. \begin{aligned} \bar{u}_1 = & \frac{G_1}{2} \bar{y}^2 - \left(\frac{\beta_1}{h} + \frac{G_1}{2} \bar{h} \right) \bar{y} + \beta_1 \\ G_1 = & \frac{6 \beta_1}{h^2} - \frac{12}{h^3} Q_x^{(1)}, \quad Q_x^{(1)} = \frac{3}{4} \frac{\bar{h}_1^2 + \bar{h}_2^2}{\bar{h}_1^2 + \bar{h}_1 \bar{h}_2 + \bar{h}_2^2} \beta_1 \end{aligned} \right\} \quad (28)$$

总支撑力

$$W_1 = 6 L^2 \eta_0 U D \beta_1 \left[-\frac{\ln H}{2 \bar{h}_1^2} + \frac{2}{3} \frac{\ln H}{\bar{h}_1^3} + \frac{1}{4} \frac{H^2 - 1}{\bar{h}_1^2} + \frac{2}{9} \frac{1 - H^3}{\bar{h}_1^3} \right] \quad (29)$$

和总摩阻力

$$R_1 = L \eta_0 U D \beta_1 \left[4 \frac{1 - H}{\bar{h}_1} + 3 Q_1^{(0)} \frac{H^2 - 1}{\bar{h}_1^2} \right] \quad (30)$$

二阶解:

$$\left. \begin{aligned} \bar{u}_2 &= \frac{G_2}{2} \bar{y}^2 + 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) G_0 \left\{ \frac{G_0^2}{12} \bar{y}^4 + \frac{\bar{y}^2}{2 \bar{h}^2} + \frac{G_0}{2} \bar{y}^2 + \frac{G_0}{8} \bar{h}^2 \bar{y}^2 - \frac{G_0}{3} \frac{\bar{y}^3}{\bar{h}} - \frac{G_0^2}{6} \bar{h} \bar{y}^3 \right\} - \left(\frac{1}{\bar{h}} + \frac{\bar{h}}{2} G_2 \right. \\ &\quad \left. + 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) G_0 \left(\frac{G_0^2}{12} \bar{h}^3 + \frac{1}{2 \bar{h}} + \frac{G_0}{2} \bar{h} + \frac{G_0^2}{8} \bar{h}^3 - \frac{G_0}{3} \bar{h} - \frac{G_0^2}{6} \bar{h}^3 \right) \right) \bar{y} + \beta_2 \\ G_2 &= -\frac{12}{\bar{h}^3} Q_1^{(2)} - \frac{18}{\bar{h}^2} - 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) \left\{ \frac{54}{5 \bar{h}^4} + \frac{324}{5 \bar{h}^5} Q_1^{(0)} + \frac{648}{5 \bar{h}^6} Q_1^{(0)^2} - \frac{432}{5 \bar{h}^7} Q_1^{(0)^3} + \frac{6}{\bar{h}^4} - \frac{12}{\bar{h}^2} Q_1^{(0)} \right\} \\ Q_1^{(2)} &= \frac{3 \bar{h}_1^3 \bar{h}_2^3}{4 (\bar{h}_1^3 - \bar{h}_2^3)} \left\{ 3 \left(\frac{1}{\bar{h}_1^3} - \frac{1}{\bar{h}_2^3} \right) + \bar{\mu}_0 (1 - \bar{\lambda}_2) \left[\frac{27}{16} \left(\frac{1}{\bar{h}_1^4} - \frac{1}{\bar{h}_2^4} \right) + \frac{324}{25} Q_1^{(0)} \left(\frac{1}{\bar{h}_2^5} - \frac{1}{\bar{h}_1^5} \right) \right. \right. \\ &\quad \left. \left. + \frac{108}{5} Q_1^{(0)^2} \left(\frac{1}{\bar{h}_1^6} - \frac{1}{\bar{h}_2^6} \right) + \frac{432}{35} Q_1^{(0)} \left(\frac{1}{\bar{h}_2^7} - \frac{1}{\bar{h}_1^7} \right) + \frac{3}{2} \left(\frac{1}{\bar{h}_1^4} - \frac{1}{\bar{h}_2^4} \right) + \frac{12}{5} Q_1^{(0)} \left(\frac{1}{\bar{h}_2^5} - \frac{1}{\bar{h}_1^5} \right) \right] \right\} \end{aligned} \right\} \quad (31)$$

总支撑力

$$\begin{aligned} W_2 &= -3 L^2 \eta_0 U D^2 \left\{ \frac{4}{3 \bar{h}_1^3} Q_1^{(2)} (H^3 - 1 - \ln H) + \frac{3 \beta_2}{\bar{h}_1^2} \left(-\ln H + \frac{H^2 - 1}{2} \right) \right. \\ &\quad \left. + \bar{\mu}_0 (1 - \bar{\lambda}_2) \left[-\frac{21}{5} \frac{\ln H}{\bar{h}_1^4} + \frac{21}{20} \frac{H^4 - 1}{\bar{h}_1^4} + \frac{384}{125} Q_1^{(0)} \frac{1 - H^5}{\bar{h}_1^5} + \frac{384}{25} Q_1^{(0)} \frac{\ln H}{\bar{h}_1^5} \right. \right. \\ &\quad \left. \left. - \frac{108}{5} Q_1^{(0)} \frac{\ln H}{\bar{h}_1^6} + \frac{18}{5} Q_1^{(0)^2} \frac{H^6 - 1}{\bar{h}_1^6} + \frac{432}{105} Q_1^{(0)^3} \frac{1 - H^7}{\bar{h}_1^7} + \frac{432}{35} Q_1^{(0)^3} \frac{\ln H}{\bar{h}_1^7} \right] \right\} \end{aligned} \quad (32)$$

和总摩阻力

$$\begin{aligned} R_2 &= L \eta_0 U D^2 \left\{ 10 \frac{1 - H}{\bar{h}_1} + 3 Q_1^{(2)} \frac{1 - H^2}{\bar{h}_1^2} \beta_2 + 3 \bar{\mu}_0 (1 - \bar{\lambda}_2) \left[\frac{76}{45} \frac{1 - H^3}{\bar{h}_1^3} \right. \right. \\ &\quad \left. \left. + \frac{48}{5} Q_1^{(0)} \frac{1 - H^4}{\bar{h}_1^4} + \frac{264}{25} Q_1^{(0)^2} \frac{1 - H^5}{\bar{h}_1^5} - \frac{36}{5} Q_1^{(0)^3} \frac{1 - H^6}{\bar{h}_1^6} \right] \right\} \end{aligned} \quad (33)$$

四、算例与结论

据文献[3]报导, W. M. Lai, S. C. Kuei 和 V. C. Mow 等人用普通青年人、老年人和骨关节炎病人的关节液进行实验, 测出本构方程(2)中的物质参量如下

	η_0 (泊)	λ_1 (秒)	λ_2 (秒)	μ_0 (秒)
普通青年人	8.20	4.2×10^{-1}	3.3×10^{-3}	8.3 ¹⁾
老年人	2.16	2.2×10^{-1}	1.6×10^{-2}	5.4 ¹⁾
骨关节炎病人	0.25	8.8×10^{-2}	1.1×10^{-2}	1.03

1) 文献[3]中没有给出这两个 μ_0 值, 我们试取不同的 μ_0 值进行计算, 得到的总支撑力与总的摩阻力在允许误差范围内都相等, 故这里只列出其中两值, 从这里亦可看出, 非线性粘度参量 μ_0 对于该情形下的总支撑力与总摩阻力的影响甚微。

对于楔形滑缝

	W_0 (牛顿)	W_2 (牛顿)	R_0 (达因)	R_2 (达因)
普通青年人	105.821	1.305	-4940.105	336.207
老年人	102.185	-0.7×10^{-5}	-1301.301	-61.109
骨关节病人	101.431	-0.16×10^{-5}	-150.613	-69.513

从 (20)、(21) 式和 (29)、(30) 式可以看到, 当 $\beta_1 = \beta_2 = 0$ 时, $W_1 = R_1 = 0$, 而从 (23)、(24) 式和 (32)、(33) 式可以看到, 当 $\beta_1 = \beta_2 = 0$ 时, $W_2 \neq 0$, $R_2 \neq 0$. 因而当关节的下部仅以速度 U 滑动时, 即与牛顿流的润滑边界条件相同时, 得到下列计算结果

对于指数形滑缝

	W_0 (牛顿)	W_2 (牛顿)	R_0 (达因)	R_2 (达因)
普通青年人	101.299	0.00198	-851.15	0.218
老年人	101.291	-0.00295	-224.21	-0.0101
骨关节病人	101.293	-0.512	-25.91	-14.105

表中 W_0 与 R_0 相应于牛顿流产生的支撑力与摩阻力, W_2 与 R_2 是非牛顿特性产生的支撑力与摩阻力. 对于老年人和骨关节病人, W_2 为负值, 从物理意义上看, 这两类人的关节液的非牛顿润滑效应是减小牛顿流在润滑时所产生的支撑力; R_2 与 R_0 同号, 表示这两类人的关节液的非牛顿润滑效应是进一步增大牛顿流在润滑时所产生的摩阻力. 由于此时 W_1 与 R_1 都为零, 从而总支撑力 $W = W_0 + W_2$, 总摩阻力 $R = R_0 + R_2$, 从上面计算结果可以看到, 无论是对于楔形滑缝, 还是对于指数形滑缝, 青年人的关节液比起老年人和骨关节病人的关节液, 有较好的润滑效果, 即可以增大总支撑力和减小总摩阻力, 对于老年人和骨关节病人, 他们的关节液在关节运动时将引起总支撑力减小和总摩阻力增大. 特别是对于骨关节病人, 其摩阻力将增大为牛顿流摩阻力的 150% 以上, 而且指数形滑缝产生的这些效应比楔形滑缝所产生的类似效应更显著一些.

当取 $\beta_1 = 1$ 时, 计算到一阶近似的结果为

对于楔形滑缝

	W_0 (牛顿)	W_1 (牛顿)	R_0 (达因)	R_1 (达因)
普通青年人	105.821	1.902	-4940.105	-2074.841
老年人	102.185	0.262	-1301.301	-286.285
骨关节病人	101.431	0.012	-150.613	-13.254

对于指数形滑缝

	W_0 (牛顿)	W_1 (牛顿)	R_0 (达因)	R_1 (达因)
普通青年人	101.299	0.002678	-851.16	-357.48
老年人	101.291	0.00037	-224.21	-49.33
骨关节病人	101.293	0.0000171	-25.91	-2.28

从这些结果可以看到, 在一阶近似下, 当边界条件与牛顿流的润滑边界条件不同时, 对于楔形滑缝和指数形滑缝, 对所研究三类人的关节总支撑力 $W = W_0 + W_1$ 和总摩阻力 $R = R_0 + R_1$ 都是增加的, 这与我们在文献 [2] 中得到的结果类似.

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FLOW OF SYNOVIAL FLUID REPRESENTED AS AN OLDROYD FLUID BETWEEN JOINTS

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ABSTRACT Mow and Lai have characterised synovial fluid as a 4 - constant Oldroyd fluid. We have examined the flow of such a fluid (a) between two plane surfaces inclined to each other and the lower plane moving with a constant velocity, (b) between one plane surface moving with a constant velocity and another fixed surface whose shape is given by an exponential function. The above flow may be considered to be approximately the flow between joints.

Using the values of the parameters of the fluid given by Mow and Lai we have calculated the total normal force and the total resistance on the fixed surface.

KEY WORDS oldroyd fluid, synovial fluid, slider bearing gap, exponential gap, knee joint, lubrication effect, perturbation method.